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(54) **RATCHET WHEEL FOR FISHING REEL**

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(57) **ABSTRACT**

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(52) **U.S. Cl.**

CPC **A01K 89/0155** (2013.01); **A01K 89/01** (2013.01); **A01K 89/015** (2013.01); **Y10T 74/19679** (2015.01)

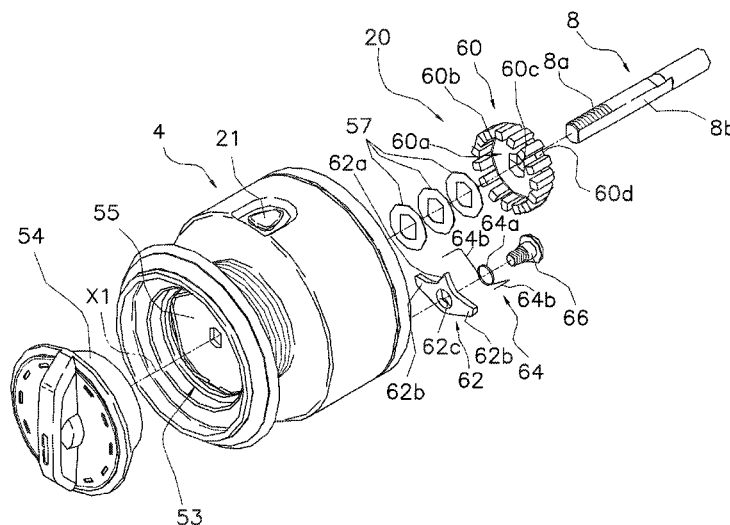
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See application file for complete search history.

A ratchet wheel is provided for a fishing reel (e.g., a spinning reel or a dual-bearing reel). The ratchet wheel is mounted onto a first component of the fishing reel. The ratchet wheel being engagable with a claw member pivotally mounted to a second component of the fishing reel. The second component is configured to be rotatable relatively to the first component. The ratchet wheel has a mounting part and a plurality of teeth. The mounting part is configured to be mounted onto the first component. The teeth are radially disposed on one of an outer peripheral portion of the mounting part and an inner peripheral portion of the mounting part. The teeth extend from the one of the outer and inner peripheral portions. The teeth are configured such that two adjacent ones of the teeth selectively engage the claw member.

13 Claims, 9 Drawing Sheets



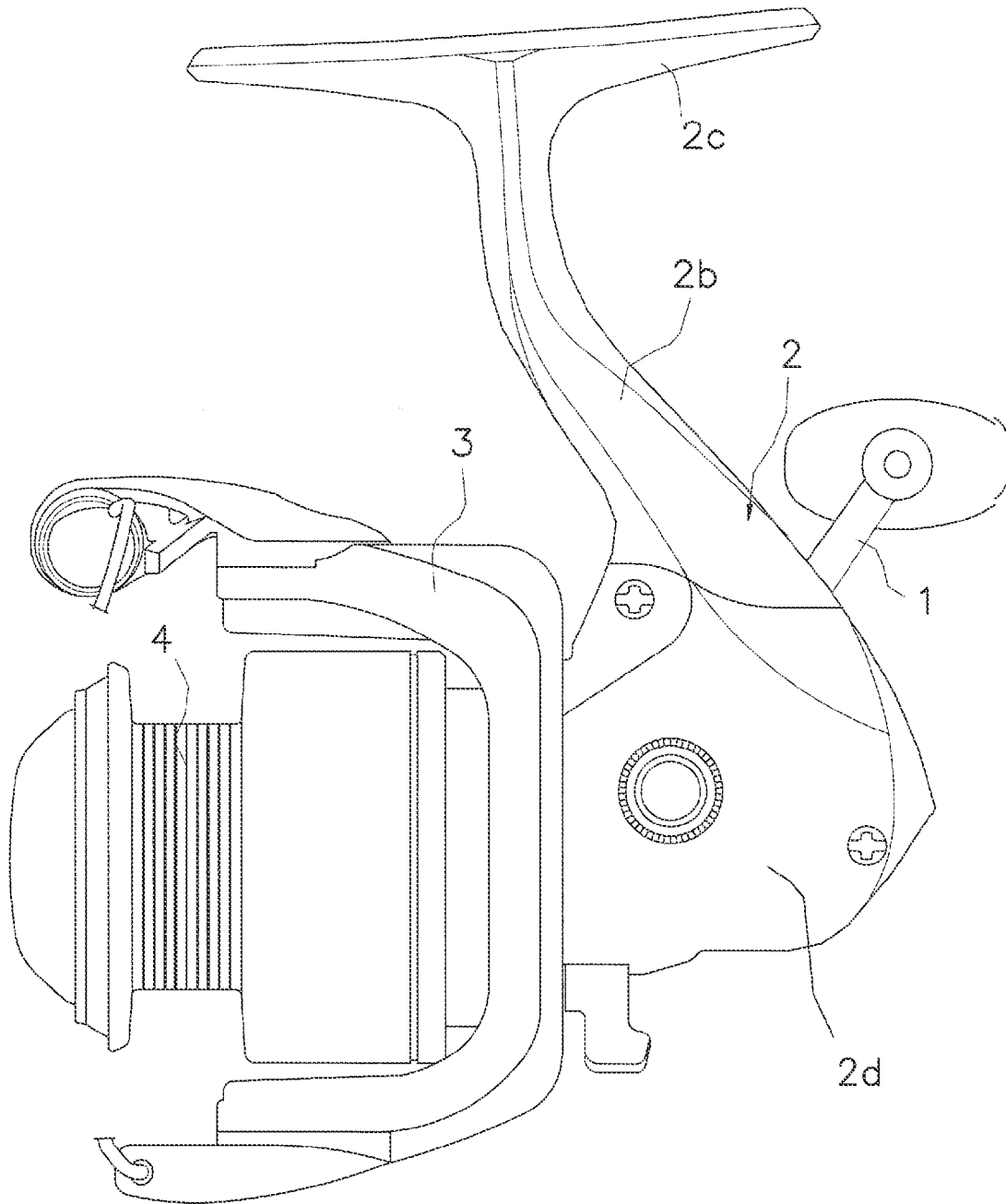


FIG. 1

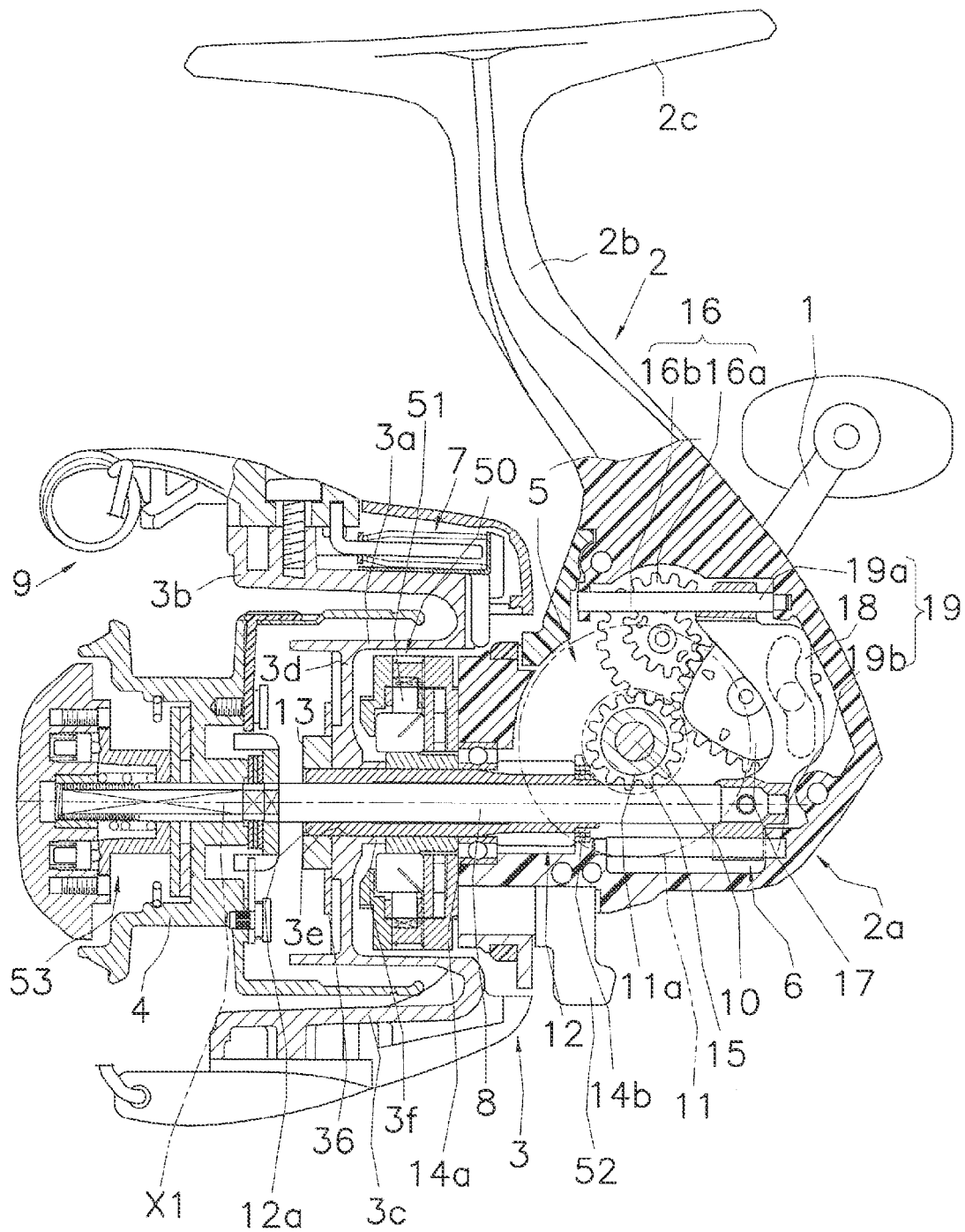


FIG. 2

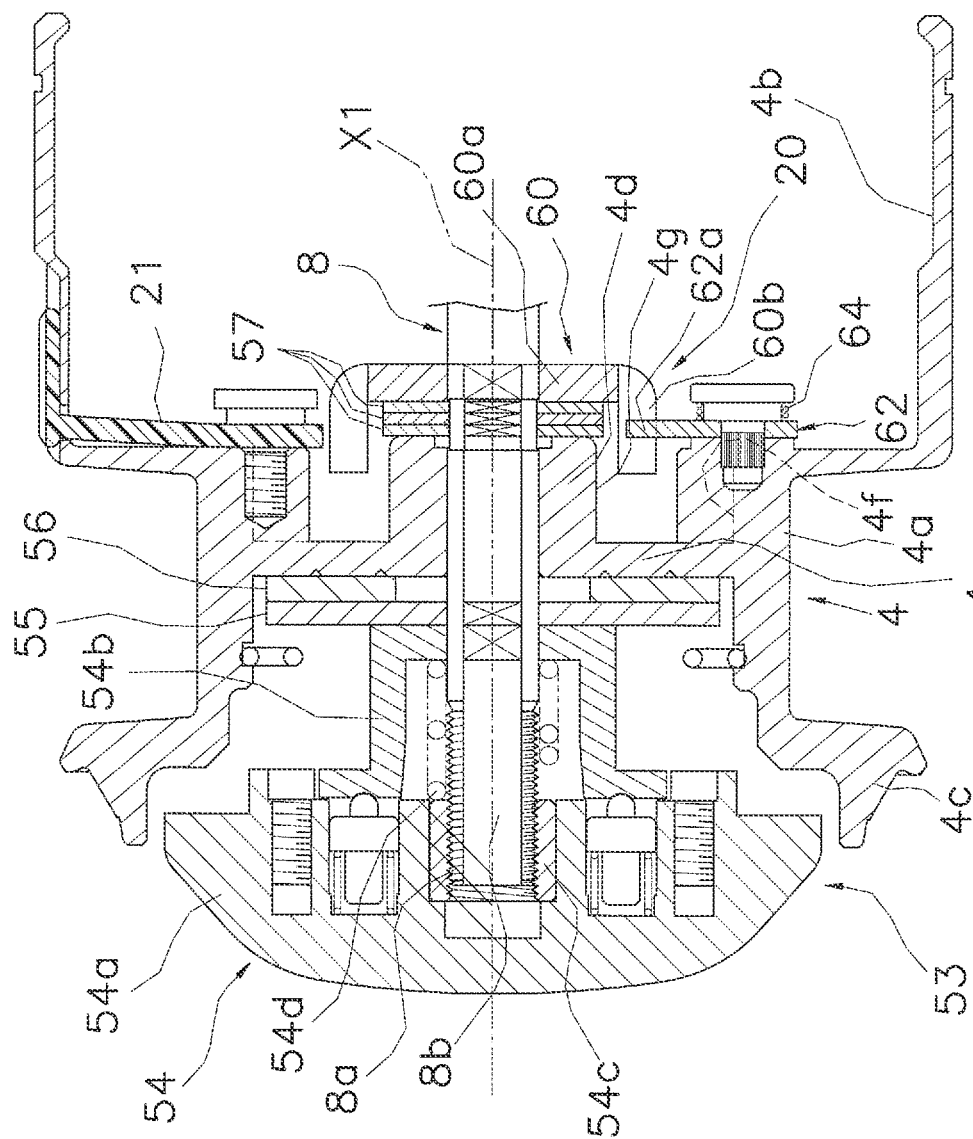
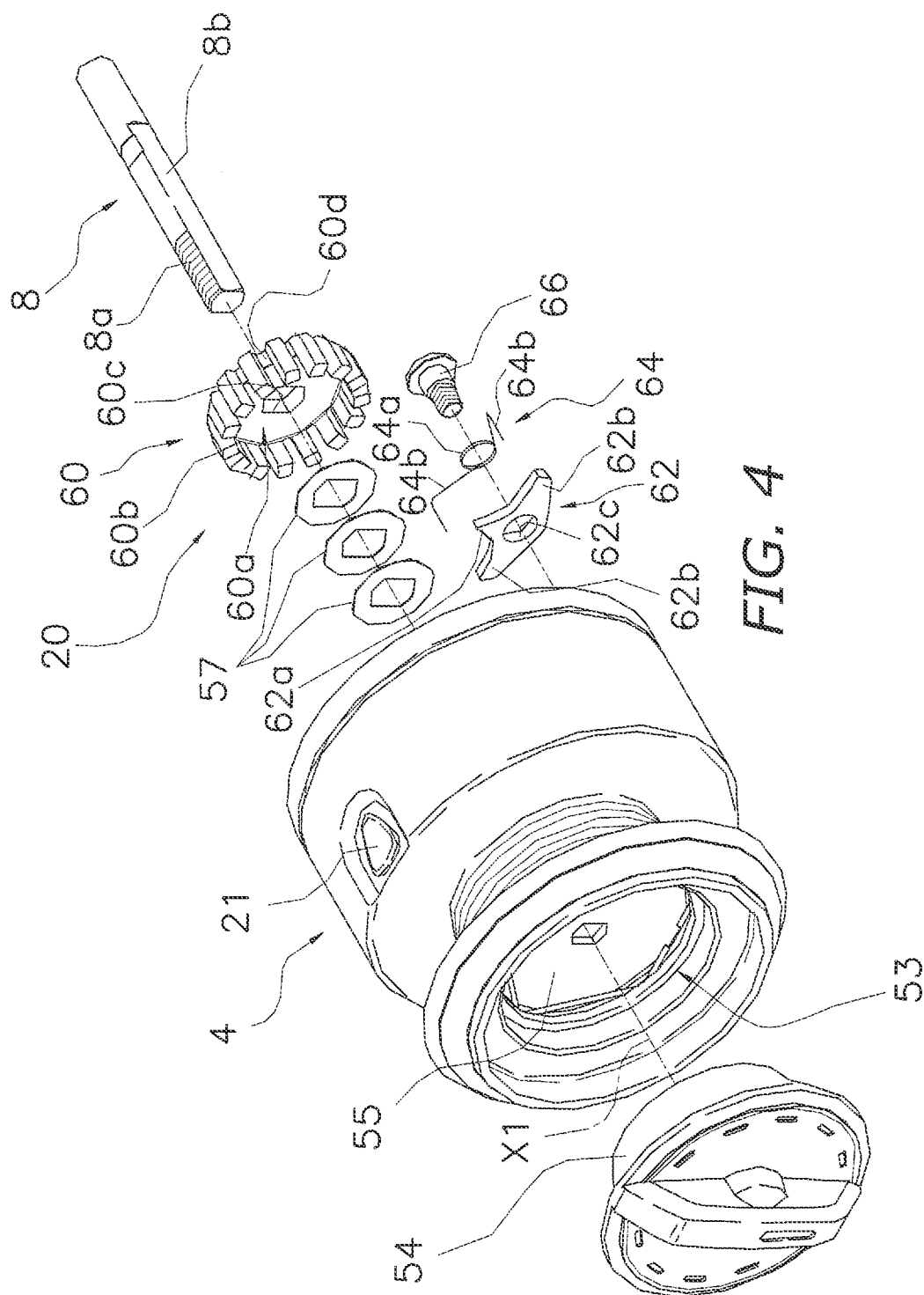


FIG 3



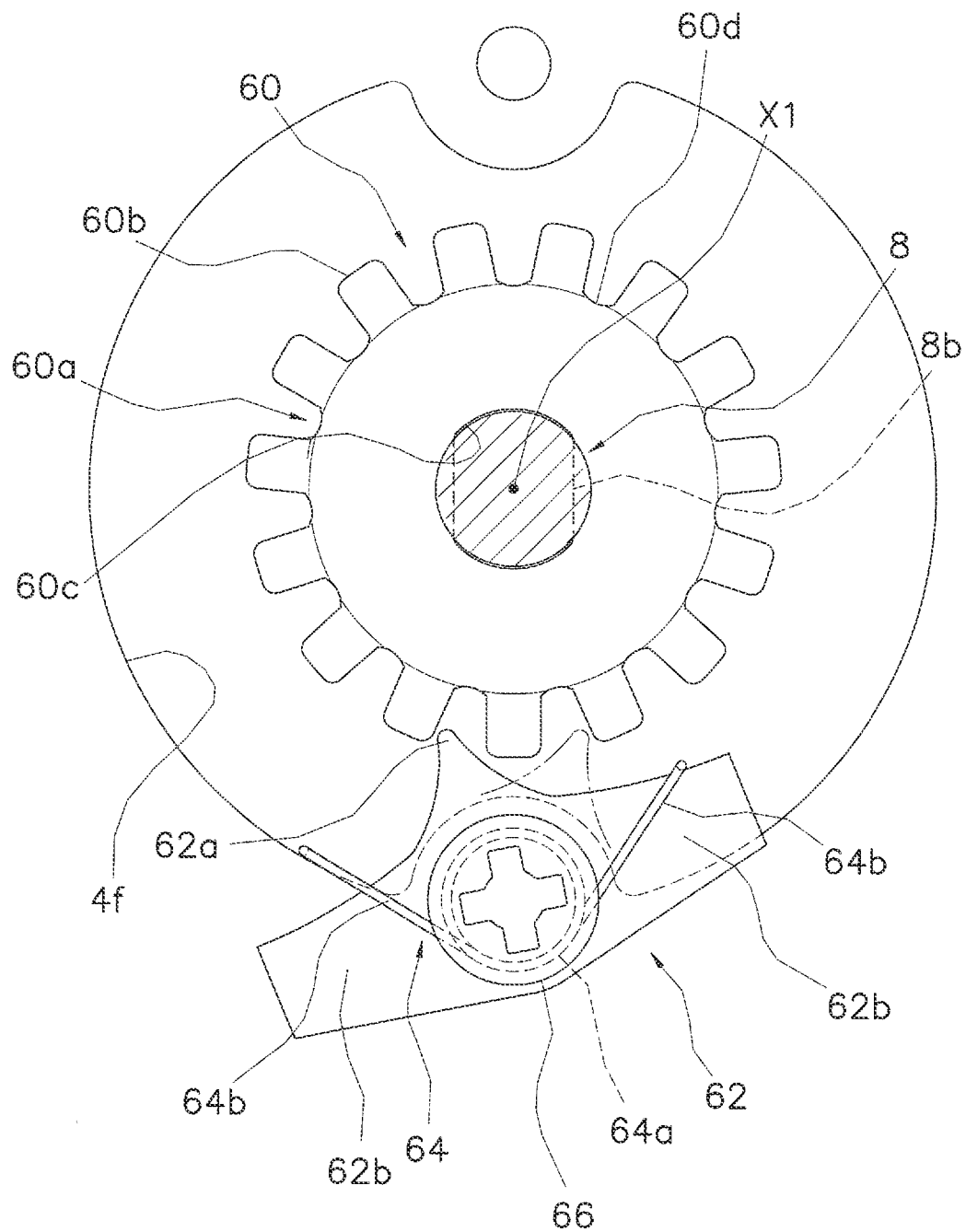


FIG. 5

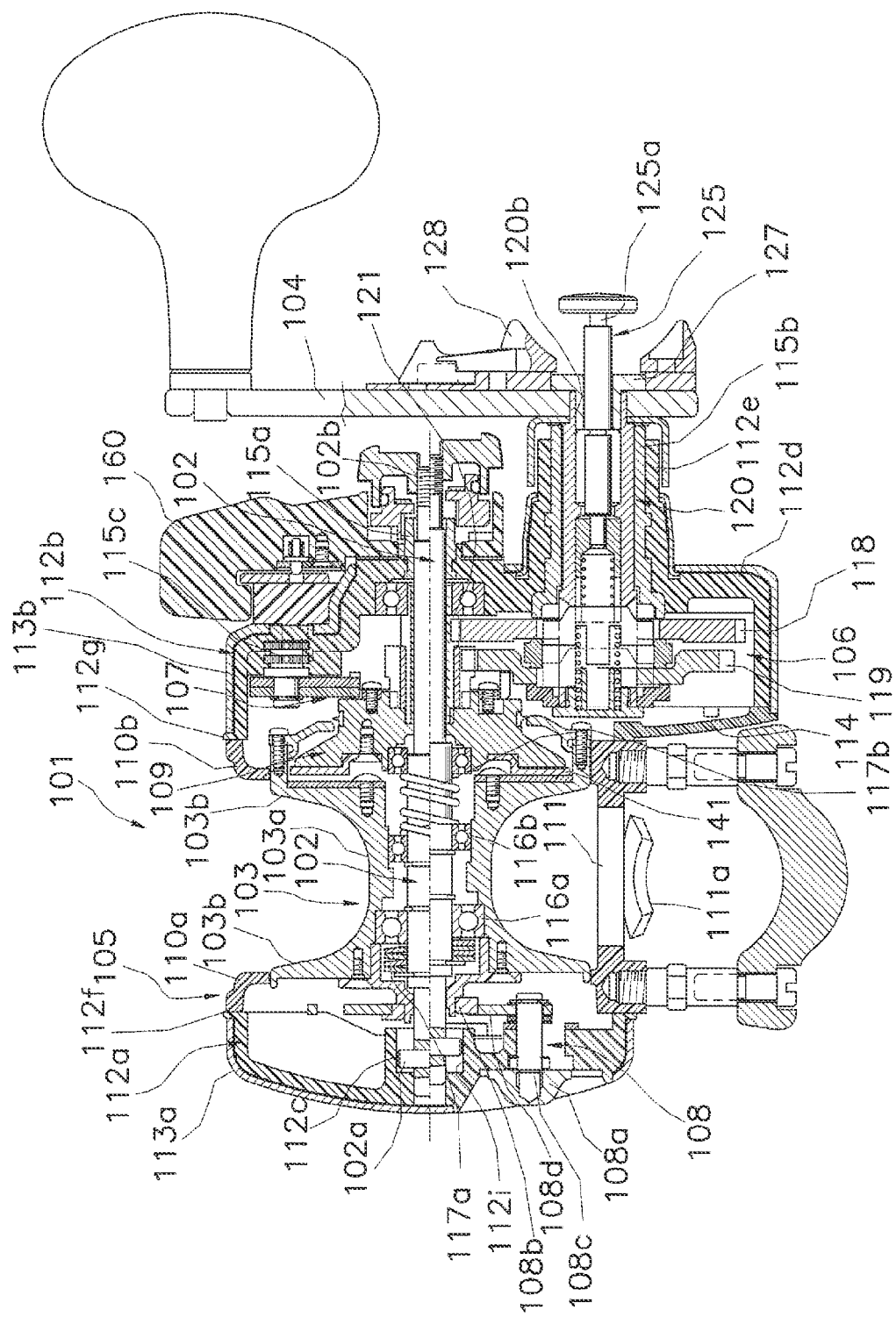


FIG. 6

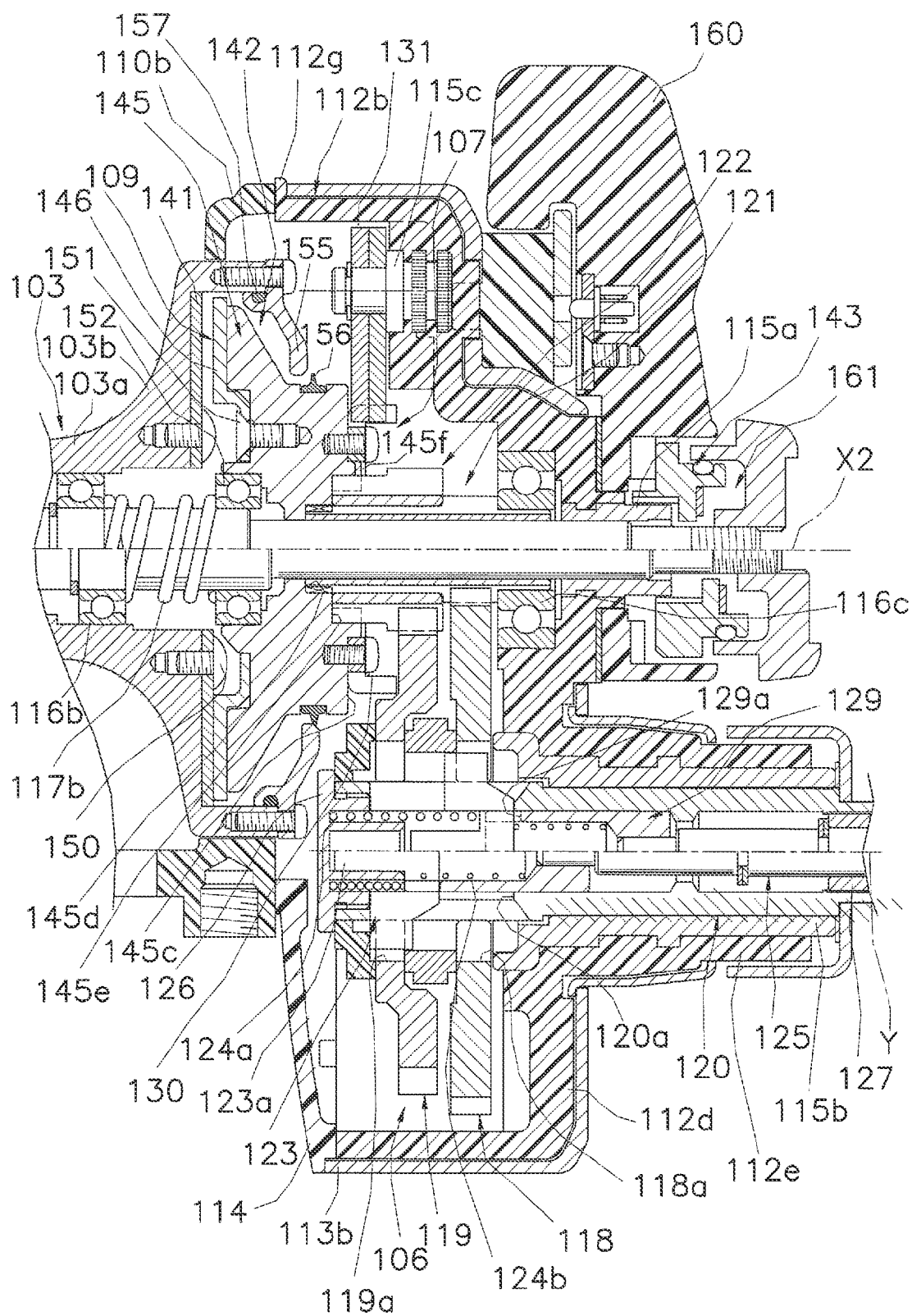


FIG. 7

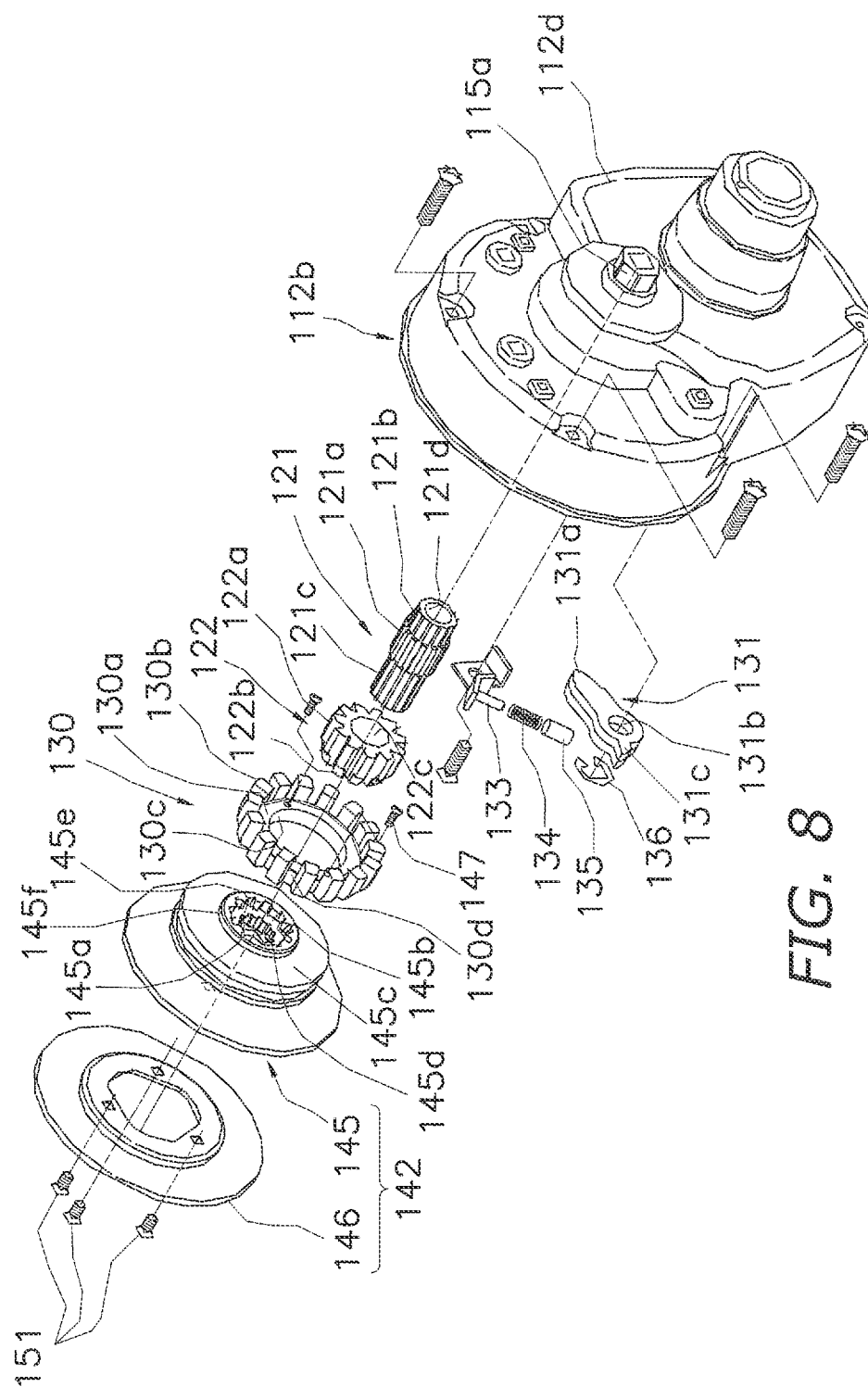


FIG. 8

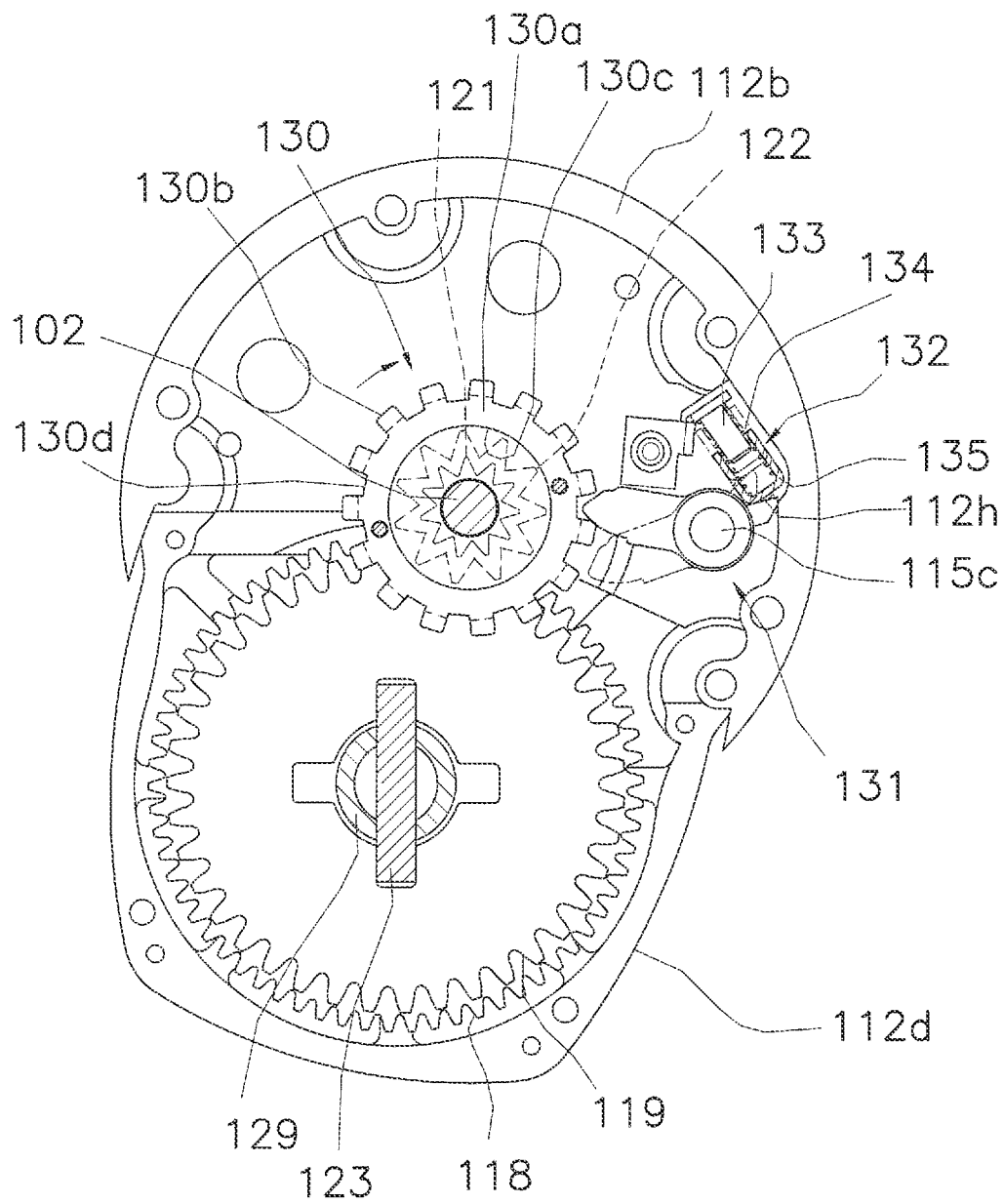


FIG. 9

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RATCHET WHEEL FOR FISHING REEL**CROSS-REFERENCE TO RELATED APPLICATIONS**

This application claims priority to Japanese Patent Application No. 2013-053357 filed on Mar. 15, 2013, the entirety of which is hereby incorporated herein by reference in its entirety.

BACKGROUND OF THE INVENTION**1. Field of the Invention**

The present invention generally relates to a ratchet wheel. More particularly, the present invention relates to a ratchet wheel for a fishing reel, which is attached to a first component for the fishing reel, and is engaged with a claw member, which is pivotally mounted to a second component that is configured to be rotatable relatively to the first component.

2. Background Art

In a dual-bearing reel as one type of fishing reels, ratchet wheels are used for an anti-reverse mechanism configured to prevent reverse rotation of a spool and a spool sound producing mechanism configured to produce a sound in conjunction with rotation of the spool. On the other hand, in a spinning reel as another type of fishing reels, a ratchet wheel is used for a sound producing mechanism configured to produce a sound in conjunction with rotation of a spool. Such a well-known ratchet wheel is a plate-shaped member, and has a plurality of teeth that are disposed on the outer peripheral surface of the ratchet wheel while being circumferentially aligned at intervals (see e.g., Japanese Laid-Open Patent Application Publication No. JP-A-2006-087301).

The conventional ratchet wheel is formed by a plate-shaped member, and therefore, the axial thickness thereof is small. Where such a ratchet wheel has a small thickness, there are chances of producing a trouble that a sound producing mechanism cannot normally produce a sound when the ratchet wheel and a claw member are axially displaced from each other. On the other hand, there are chances of producing a trouble that the claw member is disengaged from the ratchet wheel in the anti-reverse mechanism and this makes it impossible to prevent reverse rotation of the spool. Therefore, the well-known claw member is axially elongated to cope with such troubles. For example, to axially elongate the claw member, the claw member is increased in its thickness, or alternatively, is formed in a shape bent along the axial direction. However, when the claw member is bent along the axial direction, the strength of the claw member is degraded. Therefore, such claw member can be used for the spool sound producing mechanism but cannot be used for the anti-reverse mechanism.

It is an object of the present invention to inhibit occurrence of a trouble in a ratchet wheel without elongating the axial length of a claw member even when the ratchet wheel and the claw member are displaced from each other.

SUMMARY

A ratchet wheel for a fishing reel according to the present invention is mounted to a first component for the fishing reel. The ratchet wheel is engaged with a claw portion of a claw member pivotally mounted to a second component configured to be rotatable relatively to the first component. The ratchet wheel for a fishing reel includes a mounting part and a plurality of teeth. The mounting part is mountable to the first component. The teeth are radially disposed on either an outer

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peripheral portion or an inner peripheral portion of the mounting part. The teeth are bent from either the outer peripheral portion or the inner peripheral portion, and extend along a rotary axis of either the first component or the second component. The teeth allow the claw portion of the claw member to be engaged with any adjacent two thereof.

In the ratchet wheel for a fishing reel, the claw portion of the claw member is engaged with any adjacent two of the teeth. The teeth extend along the rotary axis. Therefore, even when the ratchet wheel and the claw member are displaced from each other in the rotary axial direction, the claw portion of the claw member is allowed to be engaged with the teeth. Here, the teeth of the ratchet wheel are formed so as to extend from the mounting part along the direction of the rotary axis. Therefore, the length of the teeth along the axial direction becomes larger than the thickness of the mounting part. Accordingly, troubles become unlikely to be caused without elongating the axial length of the claw member even when the ratchet wheel and the claw member are displaced from each other.

The mounting part and the teeth can be integrally formed by a plate-shaped member and the teeth can be bent by press work. In this case, the ratchet wheel can be easily formed, for example, by producing a circular part to be obtained as the mounting part and radial portions to be obtained as the teeth disposed on the outer (or inner) peripheral portion of the circular part to be obtained as the mounting part by punch work, and then, by bending the radial portions to be obtained as the teeth by press work.

The teeth can be bent from the outer peripheral portion of the mounting part. In this case, the teeth are formed on the outer peripheral portion of the mounting part. Therefore, the portions to be obtained as the teeth extend such that the gaps therebetween gradually extend radially outward. Therefore, no limitation is imposed on the length of the teeth along the axial direction. Thus, the mounting part and the teeth can be easily formed by punching.

The teeth can be bent from the inner peripheral portion of the mounting part. In this case, a limitation is imposed on the length of the teeth along the axial direction when the portions to be obtained as the teeth are formed by punching. However, the ratchet wheel can be formed with a large diameter.

The first component can be a spool shaft for a spinning reel of the fishing reel, and the second component can be a spool rotatably mounted onto the spool shaft. Further, the mounting part can be non-rotatably mounted onto the spool shaft and can be used for a spool sound producing mechanism configured to produce a sound in conjunction with rotation of the spool. In this case, troubles become unlikely to be caused in the spool sound producing mechanism of the spinning reel, even when back-and-front adjustment is performed for the spool.

The first component can be a rotary member configured to be rotated in conjunction with rotation of a handle for a dual-bearing reel of the fishing reel, and the second component can be a reel unit for the dual-bearing reel. Further, the mounting part can be mounted to the rotary member in a unitarily rotatable state, and can be used for an anti-reverse mechanism configured to prevent the rotary member from rotating in one direction. In this case, troubles become unlikely to be caused in the anti-reverse mechanism of a dual-bearing reel, even when the thickness of the claw member is increased in accordance with drag force.

Advantageous Effects of Invention

According to the present invention, the teeth of the ratchet wheel are formed so as to extend from the mounting part

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along the axial direction of the rotary axis. Therefore, the length of the teeth along the axial direction becomes larger than the thickness of the mounting part. Accordingly, troubles become unlikely to be caused without elongating the axial length of the claw member even when the ratchet wheel and the claw member are displaced from each other.

BRIEF DESCRIPTION OF THE DRAWINGS

Referring now to the attached drawings which form a part of this original disclosure:

FIG. 1 is a side elevational view of a spinning reel employing a ratchet wheel in accordance with a first exemplary embodiment;

FIG. 2 is a cross-sectional view of the spinning reel illustrated in FIG. 1;

FIG. 3 is a cross-sectional view of a spool and the related part thereof in the spinning reel illustrated in FIGS. 1 and 2;

FIG. 4 is an exploded perspective view of the spool and the vicinity thereof;

FIG. 5 is a front view of a spool sound producing mechanism;

FIG. 6 is a cross-sectional view of a dual-bearing reel employing a ratchet wheel in accordance with a second exemplary embodiment;

FIG. 7 is an enlarged cross-sectional view of the dual-bearing reel on the handle-side part thereof;

FIG. 8 is an exploded perspective view of an anti-reverse mechanism; and

FIG. 9 is a partial cross-sectional view of the anti-reverse mechanism seen from the inside thereof.

DETAILED DESCRIPTION OF THE EMBODIMENTS

First Exemplary Embodiment

Entire Structure

In FIG. 1, a spinning reel employing a first exemplary embodiment of the present invention is of a front drag type configured to wind a fishing line about an axis X1 arranged along the longitudinal direction of a fishing rod. The spinning reel includes a reel unit 2, a rotor 3 and a spool 4. The reel unit 2 includes a handle 1. The rotor 3 is supported at the front of the reel unit 2, while being rotatable about the axis X1. The spool 4 is disposed at the front of the rotor 3, and is configured to wind the fishing line thereon.

Reel Unit

The reel unit 2 is made of, for example, metal or synthetic resin. As illustrated in FIGS. 1 and 2, the reel unit 2 has an attachment portion 2c, a reel body 2a (see FIG. 2) and a leg portion 2b. The attachment portion 2c is elongated in a back-and-front direction of the spinning reel. The attachment portion 2c is attached to the fishing rod. The reel body 2a is disposed away from the attachment portion 2c at a predetermined distance. The leg portion 2b couples the attachment portion 2c and the reel body 2a. The reel body 2a includes a mechanism attachment space in the interior thereof, and is laterally opened. The opened part of the reel body 2a is covered with a lid member 2d.

A rotor drive mechanism 5 and an oscillating mechanism 6 are disposed in the interior of the reel body 2a. The rotor drive mechanism 5 is configured to rotate the rotor 3. The oscillating mechanism 6 is configured to reciprocate the spool 4 back and forth in conjunction with rotation of the handle 1 in the back-and-front direction of the spinning reel.

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Rotor

The rotor 3 is made of, for example, synthetic resin or metal, and is rotatably supported by the reel unit 2. The rotor 3 has a cylindrical portion 3a, a first arm portion 3b and a second arm portion 3c. The first and second arm portions 3b and 3c are disposed laterally to the cylindrical portion 3a, while being opposed to each other. Further, the cylindrical portion 3a has a boss portion 3f in the center part of a front wall 3d thereof. The boss portion 3f has a through hole 3e. A spool shaft 8 and a pinion gear 12 (to be described) extend through the through hole 3e. A bail arm 8 is mounted to the tip ends of the first arm portion 3b and the second arm portion 3c. The bail arm 8 is pivotally arranged between a fishing-line winding position and a fishing-line releasing position. The fishing line is guided onto the spool 4 by the bail arm 9.

The bail arm 9 is configured to be returned to the fishing-line winding position from the fishing-line releasing position by a bail flipping mechanism 7 that is mounted to the first arm portion 3b in conjunction with rotation of the rotor 3 in the fishing-line winding direction. The rotor 3 is prevented from being rotated in the fishing-line winding direction by an anti-reverse mechanism 50. The anti-reverse mechanism 50 includes a one-way clutch 51 and a switch lever 52. The one-way clutch 51 is of a roller type and can be switched between a reverse rotation prevented state and a reverse rotation allowed state. The switch lever 52 is configured to switch the one-way clutch 51 between the reverse rotation prevented state and the reverse rotation allowed state.

Rotor Drive Mechanism

The rotor drive mechanism 5 includes a drive gear 11 and the pinion gear 12. The drive gear 11 has a drive gear shaft 11a that is configured to be rotated together with a drive shaft 10 onto which the handle 1 is fixed. The pinion gear 12 is meshed with the drive gear 11. The drive gear shaft 11a is rotatably supported by the reel unit 2. The pinion gear 12 is a tubular member that the spool shaft 8 extends through the center thereof. The front portion of the pinion gear 12 extends toward the spool 4, while extending through the through hole 3e of the rotor 3. The rotor 3 is non-rotatably fixed to the pinion gear 12 at the front portion 12a by a nut 13. The pinion gear 12 is rotatably supported by the reel unit 2 through a pair of bearings 14a and 14b. Specifically, the intermediate portion of the pinion gear 12 is supported by the bearing 14a, while the rear portion of the pinion gear 12 is supported by the bearing 14b. The nut 13 is prevented from being loosened by a retainer 36. The retainer 36 is fixed by screws screwed into screw holes bored in the front wall 3d.

Spool and Spool Shaft

As illustrated in FIG. 2, the spool 4 is a member made of, for example, metal or synthetic resin, and is disposed between the first and second arm portions 3b and 3c of the rotor 3. The spool 4 is rotatably mounted to the tip end of the spool shaft 8 disposed along the axis X1 in the back-and-front direction through a drag mechanism 53. As illustrated in FIGS. 3 and 4, the spool 4 has a bobbin trunk 4a, a skirt 4b and a front flange 4c. The bobbin trunk 4a is a tubular portion that the fishing line is wound onto the outer peripheral surface thereof. The skirt 4b is a large-diameter tubular portion integrally formed on the rear end of the bobbin trunk 4a. The front flange 4c is a large-diameter portion integrally formed on the front end of the bobbin trunk 4a. As illustrated in FIG. 3, the spool 4 further has a boss portion 4d and a disc portion 4e. The boss portion 4d is formed in the inner peripheral part of the spool 4 and is rotatably mounted onto the spool shaft 8. The disc portion 4e couples the boss portion 4d and the bobbin trunk 4a. Further, a tubular space is produced on the rear side of the bobbin trunk 4a. The tubular space is defined by an inner

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peripheral surface **4f** of the bobbin trunk **4a**, an outer peripheral surface **4g** of the boss portion **4d** and the disc portion **4e**. Yet further, a spool sound producing mechanism **20** and a fishing-line hook **21** are disposed on the rear surface of the bobbin trunk **4a**, while being separated at an angular interval of 180 degrees. The spool sound producing mechanism **20** is configured to produce a sound in conjunction with rotation of the spool **4**. The fishing-line hook **21** functions as a hook on which the fishing line is hooked.

As illustrated in FIGS. **3** and **4**, the spool shaft **8** is disposed along the back-and-front direction. The spool shaft **8** has a male threaded portion **8a** and a pair of anti-rotation portions **8b**. The male threaded portion **8a** is formed on the tip end of the spool shaft **8**. The anti-rotation portions **8b** are portions chamfered in parallel to each other. A slider part **18** (to be described) of the oscillating mechanism **6** is fixed to the rear end of the spool shaft **8**. The axial length of the anti-rotation portions **8b** is larger than that of the male threaded portion **8a**. Drag Mechanism

As illustrated in FIG. **3**, the drag mechanism **53** includes a drag knob **54**, a drag plate **55** and a drag washer **56**. The drag knob **54** is a member for regulating drag force. The drag plate **55** is configured to be pressed by the drag knob **54**. The drag washer **56** is disposed between the drag plate **55** and the disc portion **4e** of the spool **4**. The drag knob **54** includes a knob body **54a** and a pressing member **54b**. The pressing member **54b** is non-rotatably engaged with the spool shaft **8**. The knob body **54a** includes a nut member **54c** in the inner peripheral part thereof. The nut member **54c** is screwed onto the male threaded portion **8a** of the spool shaft **8**. The pressing member **54b** is engaged with the anti-rotation portions **8b** of the spool shaft **8**. The pressing member **54b** is thus supported by the spool shaft **8** while being non-rotatable and axially movable. Further, a coil spring **54d** is disposed between the pressing member **54b** and the nut member **54c**, while being compressed therebetween. The drag plate **55** is non-rotatably engaged with the anti-rotation portions **8b**. Further, the rear end surface of the boss portion **4d** of the spool **4** has a plurality of (e.g., three) adjuster washers **57** disposed thereon. The adjuster washers **57** serve to adjust the axial position of the spool **4**. The adjuster washers **57** are engaged with the anti-rotation portions **8b**. A ratchet wheel **60** is disposed on the rear ends of the anti-rotation portions **8b**. The ratchet wheel **60** is a ratchet wheel according to the first exemplary embodiment of the present invention, and forms a part of the spool sound producing mechanism **20**. Backward moving of the ratchet wheel **60** is restricted because of the structure that the ratchet wheel **60** is engaged with the rear ends of the anti-rotation portions **8b**. While forming a part of the spool sound producing mechanism **20** as described above, the ratchet wheel **60** also has a function of restricting backward moving of the spool **4** and actuating the drag mechanism **53**. It should be noted that in FIG. **3**, intersecting fine lines, depicted on the anti-rotation portions **8b**, indicate sections with which components are engaged.

Spool Sound Producing Mechanism

As illustrated in FIGS. **3**, **4** and **5**, the spool sound producing mechanism **20** includes the ratchet wheel **60**, a claw member **62** for producing a sound, and an urging member **64**. The ratchet wheel **60** is non-rotatably mounted onto the spool shaft **8** (e.g., a first component). The claw member **62** is pivotally attached to the spool **4** (e.g., a second component). The claw member **62** is configured to repeatedly collide with the ratchet wheel **60**. The urging member **64** urges the claw member **62**. The ratchet wheel **60** has a mounting part **60a** and teeth **60b**. The mounting part **60a** is mountable onto the rear

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ends of the anti-rotation portions **8b** of the spool shaft **8**. The teeth **60b** are disposed on an outer peripheral portion **60d** of the mounting part **60a**.

The mounting part **60a** has a non-circular hole **60c** in the center thereof. The non-circular hole **60c** is non-rotatably engaged with the anti-rotation portions **8b** of the spool shaft **8**. In the present exemplary embodiment, the non-circular hole **60c** is formed in an elliptic slot shape. However, the non-circular hole **60c** can be formed in any suitable shape (e.g., a rectangular shape) as long as it can be engaged with the parallel surfaces of the anti-rotation portions **8b**.

The teeth **60b** are radially disposed on the outer peripheral portion **60d** of the mounting part **60a**. The teeth **60b** are bent from the outer peripheral portion **60d** of the mounting part **60a**, and extend along the axis **X1** of the spool shaft **8** (an exemplary rotary axis of the first component and the second component). A claw portion **62a** (to be described) of the claw member **62** is engaged between adjacent two of the teeth **60b**. For example, the ratchet wheel **60** is formed by producing a part to be obtained as the non-circular hole **60c** and a part to be obtained as the teeth **60c** by punching of a metal plate and then bending the radially punched part to be obtained as the teeth **60b** by pressing.

The claw member **62** is pivotally attached to the rear surface of the bobbin trunk **4a** of the spool **4**. The claw member **62** is supported by a pivot support shaft **66**. The pivot support shaft **66** is made in the form of a bolt, and is screwed into the rear surface of the bobbin trunk **4a**. The claw member **62** has the claw portion **62a**, a pair of spring hooked portions **62b** and a support hole **62c**. The claw portion **62a** is disposed in the center of the claw member **62**. The spring hooked portions **62b** extend from the claw portion **62a** to the both lateral sides. The support hole **62c** is supported by the pivot support shaft **66** disposed in alignment with the claw portion **62a**.

The urging member **64** urges the claw member **62** to a neutral position. In the neutral position, the claw portion **62a** is disposed between adjacent two of the teeth **60b** of the ratchet wheel **60**. For example, the urging member **64** is made in the form of a coil spring made of a spring wire rod. The urging member **64** is mounted onto the pivot support shaft **66**. The urging member **64** has a coil portion **64a** and a pair of lock portions **64b**. The coil portion **64a** is mounted onto the pivot support shaft **66**. The lock portions **64b** extend from the coil portion **64a**. The lock portions **64b** are hooked on the spring hooked portions **62b**, respectively. When the claw member **62** is pivoted, either of the lock portions **64b** is configured to be compressed while being hooked on the inner peripheral surface **4f** of the bobbin trunk **4a** of the spool **4**. Accordingly, the claw member **62** is urged towards the neutral position.

In the spinning reel, the number of the adjuster washers **57** is increased or reduced, for example, in order to form the wound shape of the fishing line about the bobbin trunk **4a** in a desired shape. When such adjustment is performed, the axial position of the claw member **62** mounted to the spool **4** is displaced. However, the teeth **60b** of the ratchet wheel **60** has a large axial length in the present exemplary embodiment, and hence, no malfunction is caused in the spool sound producing mechanism **20** even when the positional adjustment is performed for the spool **4** and thereby the axial position of the claw member **62** is displaced as described above.

Oscillating Mechanism

As illustrated in FIG. **2**, the oscillating mechanism **6** is a mechanism configured to move back and forth the spool shaft **8** fixed to the center part of the spool **4** in order to move the spool **4** in the same direction as the spool shaft **8**. The oscillating mechanism **6** includes a drive gear part **15**, an interme-

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drate gear part 16, a cam gear part 17, the slider part 18 and a guide part 19. The drive gear part 15 is mounted onto the drive gear shaft 11a, while being integrated therewith (or separated therefrom). The intermediate gear part 16 has a large-diameter gear portion 16a and a small-diameter gear portion 16b. The large-diameter portion 16a is meshed with the drive gear part 15. The small-diameter portion 16b is meshed with the cam gear part 17. The cam gear part 17 is meshed with the small-diameter portion 16b, and is rotated in conjunction with the drive gear part 15 while the rotation speed thereof is much reduced than that of the drive gear part 15. The slider part 18 is engaged with the cam gear part 17, while being non-rotatably and immovably fixed to the rear end of the spool shaft 8. The slider part 18 is configured to be moved back and forth, while being engaged with the cam gear part 17. The guide part 19 is attached to the reel body 2a, and guides the slider part 18 such that the slider part 18 can be moved back and forth. The guide part 19 is composed of two guide shafts 19a and 19b. The guide shafts 19a and 19b are disposed one above the other at a prescribed spacing.

For example, the drive gear part 15 is a circular gear (a spur gear, a helical gear, etc.) and the number of its gear teeth is 14. The drive gear part 15 is integrally mounted onto the drive gear shaft 11a, while being disposed away from the drive gear 11 at a prescribed distance. The intermediate gear part 16 is a disc-shaped member. The intermediate gear part 16 is mounted to the reel body 2a while being rotatable about an axis arranged in parallel to the drive gear shaft 11a. The small-diameter portion 16b and the cam gear part 17 are non-circular gears. The number of gear teeth of the cam gear part 17 is larger than that of the small-diameter portion 16b. Operation and Action of Reel

In casting, the bail arm 9 is flipped over to the fishing-line releasing position, while the fishing line is hooked by the index finger of an angler. Then, the fishing rod is cast under the condition.

In winding the fishing line, the bail arm 9 is flipped over to the fishing-line winding position. When fish gets caught in a terminal tackle under the condition, the handle 1 is rotated in the fishing-line winding direction. When the handle 1 is rotated, the rotational force is transmitted to the pinion gear 12 through the drive shaft 10 and the drive gear 11. The rotational force, transmitted to the pinion gear 12, is transmitted to the rotor 3 through the front part of the pinion gear 12. Accordingly, the fishing line is wound about the spool 4. At this time, the spool 4 is reciprocated back and forth by the oscillating mechanism 6.

In trying to wind the fishing line about the spool 4 when a fish is hooked on the terminal tackle of the fishing line, the fishing line is reeled out of the spool 4 and the spool 4 is rotated in the fishing-line releasing direction if the fish pulls the fishing line with a force greater than a set drag force. When the spool 4 is rotated, the claw member 62 of the spool sound producing mechanism 20 is rotated together with the spool 4 and repeatedly collides with the ratchet wheel 60.

Accordingly, the spool sound producing mechanism 20 produces a sound. At this time, the teeth 60b are elastically deformed. Therefore, the teeth 60b are unlikely to be abraded. Further, a clear click sound is herein produced due to vibration of the teeth 60b.

Further, the spool sound producing mechanism 20 has a structure that the teeth 60b of the ratchet wheel 60 are formed so as to extend from the mounting part 60a along the axial direction of the spool shaft 8. Therefore, the length of the teeth 60b along the axial direction of the spool shaft 8 is larger than the thickness of the mounting part 60a. With the structure, troubles become unlikely to be caused without elongat-

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ing the axial length of the claw member 62 even when the ratchet wheel 60 and the claw member 62 are displaced from each other.

Second Exemplary Embodiment

Entire Structure

As illustrated in FIG. 6, a dual-bearing reel employing a second exemplary embodiment of the present invention is a medium-sized lever drag reel. The lever drag reel includes a tubular reel unit 101, a spool shaft 102, a spool 103 and a handle 104. The spool shaft 102 is mounted to the center part of the reel unit 101, while being non-rotatable and axially movable. The spool 103 is supported by the spool shaft 102, while being rotatable and axially immovable. The handle 104 is disposed laterally to the reel unit 101. Further, the lever drag reel includes a rotation transmission mechanism 106, an anti-reverse mechanism 107, a lever drag mechanism 109 in the interior of the reel unit 101. The rotation transmission mechanism 106 is configured to transmit rotation of the handle 104 to the spool 103. The anti-reverse mechanism 107 is provided for enabling the lever drag mechanism 109 to be actuated. The anti-reverse mechanism 107 is configured to prevent a drag disc 142 to be described from rotating in the fishing-line releasing direction. The lever drag mechanism 109 is configured to brake rotation of the spool 103 in the fishing-line releasing direction.

Reel Unit

The reel unit 101 includes a frame 105 made of metal. The frame 105 includes a first side plate 110a, a second side plate 110b and a plurality of coupling portions 111. The first side plate 110a and the second side plate 110b are a right and left pair of saucer-shaped members made of metal. The coupling portions 111 couple the first side plate 110a and the second side plate 110b at the front, the rear and the bottom of the frame 105. A fishing rod attachment portion 111a is integrally formed with the bottom one of the coupling portions 111 of the frame 105. The fishing rod attachment portion 111a is a portion for attaching the lever drag reel to a fishing rod.

Further, the reel unit 101 includes a first cover member 112a, a second cover member 112b, a third cover member 113a, a fourth cover member 113b and a lid member 114. The first and second cover members 112a and 112b are made of, for example, high stiffness resin (e.g., glass fiber reinforced polyamide resin). The first cover member 112a covers the outside of the first side plate 110a, while the second cover member 112b covers the outside of the second side plate 110b. The third and fourth cover members 113a and 113b are ornamental members made of, for example, light metal (e.g., aluminum alloy). The third cover member 113a covers the first cover member 112a, while the fourth cover member 113b covers the second cover member 112b. The lid member 114 is made of high stiffness resin, and covers the inner side of the second cover member 112b. Each of the first and second side plates 110a and 110b has an opening for allowing the spool 103 to pass therethrough. The first cover member 112a has a boss portion 112c formed in the inside thereof. The left end of the spool shaft 102 is supported by the boss portion 112c, while being axially movable and non-rotatable. A spool sound producing mechanism 108 is disposed inside the first cover member 112a. The spool sound producing mechanism 108 is configured to produce a sound in conjunction with rotation of the spool 103.

The spool sound producing mechanism 108 is configured to be switchable between a sound producing state and a silent state in response to an operation of a click button 108a. The

spool sound producing mechanism **108** includes a ratchet wheel **108b** and a claw member **108d**. The ratchet wheel **108b** is configured to be unitarily rotated with the spool **103**. The claw member **108d** is pivotally supported by a shaft member **108c** attached to the click button **108a**. The claw member **108d** is movable between a sound producing position and a silent position in response to an operation of the click button **108a** along the radial direction. The claw member **108d** is configured to be engaged with the ratchet wheel **108b** in the sound producing position. On the other hand, the claw member **108d** is configured to be separated away from the ratchet wheel **108b** in the silent position.

As illustrated in FIG. 7, a bulged portion **112d** is formed on the second cover member **112b** that is disposed on the side with the handle **104**, while being protruded in both a radial direction and an axially outward direction. The bulged portion **112d** is radially protruded, while being slanted slightly forwards. The rotation transmission mechanism **106** is disposed in the bulged portion **112d**. Further, a first insert member **115a** is insert-molded into the bulged portion **112d**, while being protruded radially outwards. The first insert member **115a** is a tubular member made of metal with good slidability (e.g., brass). The first insert member **115a** supports the right end of the spool shaft **102** in an axially movable state. Further, a support tubular portion **112e** is formed below the first insert member **115a**, while being protruded axially outwards. The support tubular portion **112e** supports a drive shaft **120** of the handle **104**. Further, a second insert member **115b** is insert-molded to the inside of the support tubular portion **112e**. The second insert member **115b** is a tubular member made of metal with good slidability (e.g., brass). The second insert member **115b** supports the drive shaft **120** in a rotatable state. Further, a third insert member **115c** is insert-molded into a portion of the rear part of the second cover member **112b**, i.e., the portion on which the bulged portion **112d** is not formed. The third insert member **115c** is a shaft-like member made of metal with good slidability (e.g., brass). The third insert member **115c** supports a claw member **131** (to be described) of the anti-reverse mechanism **107** in a pivotable state. It should be noted that FIG. 7 is illustrated as if the third insert member **115c** were disposed in the upper part of the second cover member **112b**. However, the third insert member **115c** is actually disposed in the rear part of the second cover member **112b** as illustrated in FIG. 8.

As illustrated in FIG. 6, the third cover member **113a** is formed along the outer surface of the first cover member **112a** so as to cover the first cover member **112a**, while the fourth cover member **113b** is formed along the outer surface of the second cover member **112b** so as to cover the second cover member **112b**. It should be noted that a protruded portion **112f** is formed on the circumferential edge of the first cover member **112a** such that the circumferential edge of the third cover member **113a** is abutted thereto, while a protruded portion **112g** is formed on the circumferential edge of the second cover member **112b** such that the circumferential edge of the fourth cover member **113b** is abutted thereto. The first and second cover members **112a** and **112b**, made of synthetic resin, are exposed to the outside only at the protruded portions **112f** and **112g**. When the third and fourth cover members **113a** and **113b** are fabricated by pressing of a thin plate made of aluminum alloy, for example, and the circumferential edges thereof are unevenly formed, such unevenly formed portions can be prevented from standing out by providing such protruded portions **112f** and **112g**.

Spool Shaft

As described above, the spool shaft **102** is supported by the boss portion **112c** of the first cover member **112a** and the first

insert member **115a** of the second cover member **112b**, while being axially movable and non-rotatable. An anti-rotation pin **102a** is attached to the left end portion of the spool shaft **102**, while radially penetrating therethrough. On the other hand, an anti-rotation slit **112i** is formed along the radial direction in the boss portion **112c** of the first cover member **112a** so as to be engaged with the anti-rotation pin **102a**.

The spool **103** is rotatably supported by the spool shaft **102** through two bearings **116a** and **116b** disposed on the outer peripheral surface of the spool shaft **102**. The bearing **116a** is urged axially inwards (i.e., rightwards in FIG. 6) by a first spring member **117a** made in the form of a disc spring, while the bearing **116b** is urged axially inwards (i.e., leftwards in FIG. 6) by a second member **117b** made in the form of a coil spring. Further, the axially inner lateral surfaces of the bearings **116a** and **116b** are restricted from inwardly moving by the spool **103** and the spool shaft **102**. The structure enables the spool shaft **102** and the spool **103** to unitarily move in the axial direction. The spool shaft **102** is configured to be axially moved together with the spool **103** by the lever drag mechanism **109**. The spool shaft **102** has a male threaded portion **102b** on the right end thereof. A component, forming a part of a moving mechanism **143** (to be described) of the lever drag mechanism **109**, is screwed onto the male threaded portion **102b**. Further, a first pinion gear **121** (to be described), which is a small-diameter member of the rotation transmission mechanism **106**, is mounted onto the outer periphery of the spool shaft **102**.

Spool

As illustrated in FIG. 6, the spool **103** has a bobbin trunk **103a** and a pair of flanges **103b**. The flanges **103b** are integrally formed on the both axial ends of the bobbin trunk **103a**. A friction disc **141** forms a part of the lever drag mechanism **109**. The friction disc **141** is fixed to the end surface of the right-side one of the flanges **103b** by screw members to be described.

Handle

The handle **104** is fixed to the protruded end of the tubular drive shaft **120** disposed below and in parallel to the spool shaft **102**. As described above, the drive shaft **120** is rotatably supported by the reel unit **101** through the second insert member **115b**. As illustrated in FIG. 7, the drive shaft **120** has a slit **120a** and a female threaded portion **120b** (see FIG. 6). The slit **120a** is formed on the base end of the drive shaft **120**, while radially penetrating therethrough. The female threaded portion **120b** is formed on the inner surface of the tip end of the drive shaft **120** in order to fix the handle **104**.

Rotation Transmission Mechanism

The rotation transmission mechanism **106** includes a speed change mechanism configured to switch handle rotation between two speed levels, i.e., a high speed level and a low speed level. As illustrated in FIG. 7, the rotation transmission mechanism **106** includes a first drive gear **118**, a second drive gear **119**, the first pinion gear **121**, a second pinion gear **122**, an engaging piece **123**, two compression springs **124a** and **124b**, and an operating shaft **125**. The first drive gear **118** is used for winding the fishing line at a high speed, while the second drive gear **119** is used for winding the fishing line at a low speed. The first and second drive gears **118** and **119** are rotatably supported by the drive shaft **120** of the handle **104**. The first pinion gear **121** and the second pinion gear **122** are rotatably mounted onto the spool shaft **102**. Simultaneously, the first pinion gear **121** is meshed with the first drive gear **118**, while the second pinion gear **122** is meshed with the second drive gear **119**. The engaging piece **123** is configured to couple either the first drive gear **118** or the second drive gear **119** to the drive shaft **120**. Accordingly, handle rotation

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is transmitted to the coupled one of the first and second drive gears **118** and **119** via the drive shaft **120**. The compression springs **124a** and **124b** position the engaging piece **123** on the both sides of the engaging piece **123**. The operating shaft **125** sets the position of the engaging piece **123**.

Each of the first and second drive gears **118** and **119** has a circular support hole (not illustrated in the figures) formed in the center part thereof, and further has two perpendicular slits **118a**, **119a** to be engaged with the engaging piece **123**. The second drive gear **119** is formed so as to be bent towards the first drive gear **118** in order to avoid interference with a ratchet wheel **130** of the anti-reverse mechanism **107**.

The first pinion gear **121** is a tubular member made of anti-corrosive metal (e.g., non-magnetic stainless alloy). The right end of the first pinion gear **121** is rotatably supported by a bearing **116c**. The bearing **116c** is attached to the bulged portion **112d**, while being disposed about the spool shaft **102**. On the other hand, the left end of the first pinion gear **121** is engaged with the drag disc **142** of the lever drag mechanism **109**, while being unitarily rotatable therewith. As illustrated in FIG. 8, the first pinion gear **121** has first gear teeth **121a**, a first bearing support portion **121b** and a first engaging and coupling portion **121c**. The first gear teeth **121a** are meshed with the first drive gear **118**. The first bearing support portion **121b** is supported by the bearing **116c**. The first engaging and coupling portion **121c** is disposed on the opposite side of the first bearing support portion **121b** through the first gear teeth **121a**. A first pass-through hole **121d** is formed in the inner peripheral part of the first pinion gear **121**. The spool shaft **102** can pass through the first pass-through hole **121d**. A clearance of roughly 0.05 to 0.3 mm is produced between the first pass-through hole **121d** and the spool shaft **102**. Therefore, the first pinion gear **121** can be smoothly rotated relatively to the spool shaft **102**.

The first pinion gear **121** is produced by cutting a blank that the first gear teeth **121a** are formed over the entire length thereof. Specifically, the both axial sides of the part to be obtained as the first gear teeth **121a** are cut such that steps in use for positioning are formed and convexo-concave portions in use for preventing rotation are left. Through the cutting process, the first bearing support portion **121b** and the first engaging and coupling portion **121c** are formed on the both axial sides of the first gear teeth **121a**. Therefore, each of the first bearing support portion **121b** and the first engaging and coupling portion **121c** has a diameter smaller than that of the first gear teeth **121a**, and is formed such that the first gear teeth **121a**, originally formed on the blank, are partially left. The first pinion gear **121** is disposed while being interposed between the drag disc **142** and an inner race of the bearing **116c**. The first pinion gear **121** is thus mounted in an axially immovable state.

The second pinion gear **122** is a tubular member made of the same material as the first pinion gear **121**. As illustrated in FIG. 7, the left end of the second pinion gear **122** is engaged with the drag disc **142**, while being unitarily rotatable therewith. As illustrated in FIG. 8, the second pinion gear **122** has second gear teeth **122a** and a second engaging and coupling portion **122b**. The second gear teeth **122a** are meshed with the second drive gear **119**. The second engaging and coupling portion **122b** is disposed adjacently to the second gear teeth **122a**. A second pass-through hole **122c** is formed in the inner peripheral part of the second pinion gear **122**. The first engaging and coupling portion **121c** of the first pinion gear **121** can pass through the second pass-through hole **122c**. A clearance of roughly 0.01 to 0.05 mm is produced between the second pass-through hole **122c** and the first engaging and coupling

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portion **121c**. Therefore, the second pinion gear **122** does not contact the first pinion gear **121**, but is substantially supported by the first pinion gear **121**.

The second pinion gear **122** is produced by cutting a blank that the second gear teeth **122a** are formed over the entire length thereof. Specifically, one axial side of the part to be obtained as the second gear teeth **122a** is cut such that a step in use for positioning is formed and convexo-concave portions in use for preventing rotation are left. Through the cutting process, the second engaging and coupling portion **122b** is formed on one side of the second gear teeth **122a**. Therefore, the second engaging and coupling portion **122b** has a diameter smaller than that of the second gear teeth **122a** and is formed such that the second gear teeth **122a**, originally formed on the blank, are partially left. The second pinion gear **122** is disposed while being interposed between the drag disc **142** and the first pinion gear **121**. The second pinion gear **122** is thus mounted in an axially immovable state.

The engaging piece **123** is non-rotatably disposed within the slit **120a** of the drive shaft **120**. The engaging piece **123** has a protruded portion **123a** in the center part thereof. When the engaging piece **123** is disposed on the side with the second drive gear **119**, the protruded portion **123a** is configured to be disposed on the inner peripheral side of a flange-shaped spring receiver **126** for receiving the compression spring **124a**. The spring receiver **126** is screwed into the base end of the drive shaft **120**.

As illustrated in FIG. 6, the operating shaft **125** is disposed inside the drive shaft **120**, while being protruded to the outside through the drive shaft **120**. The operating shaft **125** is supported by a nut **127**, while being axially movable. The nut **127** serves to fix the handle **104** to the drive shaft **120**, while the handle **104** is screwed onto the protruded end of the drive shaft **120**. The operating shaft **125** has an annular groove **125a** formed on the outwardly protruded end thereof. A slide-type stopper **128** is mounted to the handle **104**. The stopper **128** is engaged with the annular groove **125a**. Further, a spring receiver member **129** is attached onto the opposite-side end of the operating shaft **125**. The spring receiver member **129** receives the compression spring **124b**. The tip end of the operating shaft **125** is fitted into the spring receiver member **129**. The spring receiver member **129** has a slit **129a** to be engaged with the engaging piece **123**. The engaging piece **123** is pressed by the slit **129a**.

In the rotation transmission mechanism **106** structured as described above, the engaging piece **123** is disposed in the second drive gear **119** when the operating shaft **125** is inwardly pressed as depicted on the lower side of an operating axis Y in FIG. 7. The rotation of the handle **104** is thereby transmitted to the second pinion gear **122** through the second drive gear **119**. Accordingly, the spool **103** is rotated at a low speed. On the other hand, the engaging piece **123** is disposed in the first drive gear **118** when the operating shaft **125** is outwardly pulled as depicted on the upper side of the operating axis Y in FIG. 7 by sliding the slide-type stopper **128**. The rotation of the handle **104** is thereby transmitted to the first pinion gear **121** through the first drive gear **118**. Accordingly, the spool **103** is rotated at a high speed.

Structure of Anti-Reverse Mechanism

As illustrated in FIGS. 8 and 9, the anti-reverse mechanism **107** is a claw-type one-way clutch, and includes the ratchet wheel **130**, the claw member **131** and an urging member **132**. The ratchet wheel **130** has a mounting part **130a** and a plurality of teeth **130b**. The mounting part **130a** is mounted to the drag disc **142**, while being unitarily rotatable therewith. The teeth **130b** are disposed on the outer peripheral portion of the mounting part **130a**. The claw member **131** is engaged with

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the teeth **130b**. The urging member **132** urges the claw member **131** towards the teeth **130b**.

The ratchet wheel **130** is disposed on the outer peripheral side of the second pinion gear **122**. As illustrated in FIG. 8, the ratchet wheel **130** is formed by bending a metal plate. The mounting part **130a** is fixed to a lateral surface **145c** (i.e., a surface away from the spool **103**) of a disc body **145** (to be described) forming a part of the drag disc **142** by bolt members **147**. The mounting part **130a** has an axis-aligned hole **130c**. The axis-aligned hole **130c** is fitted onto an axis-aligning protrusion **145f**. The axis-aligning protrusion **145f** is circularly formed on the lateral surface **145c**, while being protruded coaxially with the axis-aligned hole **130c**. The teeth **130b** are radially disposed on an outer peripheral portion **130d** of the mounting part **130a**. The teeth **130b** are bent from the outer peripheral portion **130d** of the mounting part **130a**, and extend along an axis X2 of the spool shaft **102** (an exemplary rotary axis of the first component and the second component).

The claw member **131** is a member formed by punching a thin plate made of, for example, stainless alloy by pressing. In the present exemplary embodiment, the claw member **131** is composed of two members with the same thickness. The claw member **131** is coupled to the reel unit **101**, while being configured to pivot between an engaged position and a separated position. In the engaged position, the claw member **131** is engaged with the teeth **130b**, and thereby, the reverse rotation of the drag disc **142** is prevented. In the separated position, the claw member **131** is separated away from the teeth **130b**. The claw member **131** has a claw portion **131a**, a mounting portion **131b** and an engaging protruded portion **131c**. The claw portion **131a** is formed on the tip end of the claw member **131** so as to be capable of being engaged with the teeth **130b**. The mounting portion **131b** is made in the form of a hole, and is pivotally mounted onto the third insert member **115c**. The engaging protruded portion **131c** extends radially from the mounting portion **131b** to the opposite side of the claw portion **131a**. When the claw member **131** is set in the engaged position illustrated in FIG. 9, the engaging protruded portion **131c** serves to keep the claw member **131** in the engaged position while making contact with the inner side of the second cover member **112b**. The claw member **131** is retained with respect to the third insert member **115c** by a retainer member **136** (e.g., an E-type snap ring).

The urging member **132** includes a coil spring **134** and a pressing member **135**. The coil spring **134** is disposed on the outer peripheral side of a guide shaft **133** fixed to the second cover member **112b**. The pressing member **135** is a closed-end tubular member for pressing the engaging protruded portion **131c**. The pressing member **135** is disposed so as to cover the coil spring **134**. The pressing member **135** presses the engaging protruded portion **131c** towards a protruded portion **112h** formed on the second cover member **112b**. Accordingly, the claw member **131** is constantly urged towards the engaged position, and is configured to pivot towards the separated position only when the drag disc **142** is rotated together with the spool **103** in the fishing-line winding direction depicted with an arrow in FIG. 9.

In the anti-reverse mechanism **107** structured as described above, the ratchet wheel **130** is disposed between the second pinion gear **122** and the drag disc **142**, and the claw member **131** is configured to be engaged with the ratchet wheel **130**. Therefore, load in drag activation can be received by both of the ratchet wheel **130** and the claw member **131**. Accordingly, load does not act on the second pinion gear **122** in drag

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activation. The second gear teeth **122a** of the second pinion gear **122** can be thereby prevented from being damaged or broken.

Structure of Lever Drag Mechanism

As illustrated in FIG. 6, the lever drag mechanism **109** includes the friction disc **141**, the drag disc **142** and the moving mechanism **143**. The friction disc **141** is attached to the right end of the spool **103** in FIG. 6. The drag disc **142** is disposed in opposition to the friction disc **141**. The moving mechanism **143** is configured to axially reciprocate the spool shaft **102**.

The friction disc **141** is a washer-shaped disc member made of, for example, abrasion-resistant material (e.g., carbon graphite, fiber reinforced resin, etc.). The friction disc **141** is fixed to the outer lateral surface of the right-side flange **103b** of the spool **103** by a plurality of attachment bolts **150** circumferentially disposed at intervals.

As illustrated in FIGS. 7 and 8, the drag disc **142** includes the disc body **145** and a brake disc **146**. The drag disc **142** is an example of the first component and a rotary member. The disc body **145** is coupled to the first pinion gear **121**, the second pinion gear **122** and the ratchet wheel **130**, while being unitarily rotatable therewith. The drag disc **146** is a member made of, for example, stainless steel. The drag disc **146** is fixed to the disc body **145** by a plurality of attachment bolts **151**, while being disposed in opposition to the friction disc **141**. The drag disc **142** is prevented from rotating in the fishing-line releasing direction by the anti-reverse mechanism **107**.

The disc body **145** is a disc-shaped member made of, for example, die-cast aluminum with a high heat dissipation performance. The disc body **145** is rotatably supported by the spool shaft **102** through a bearing **152**. The drag disc **146** is fixed to a disc body **145** surface opposed to the spool **103**. The drag disc **146** and the disc body **145** surface to which the drag disc **146** is attached are formed such that the center parts thereof are recessed for avoiding interference with the attachment bolts **150** of the friction disc **141**.

The lateral surface **145c** of the disc body **145**, i.e., the surface disposed away from the spool **103**, has a first engaged and coupled portion **145a** and a second engaged and coupled portion **145b**. The first engaged and coupled portion **145a** is formed for allowing the first pinion gear **121** to be engaged therewith in a unitarily rotatable state, while the second engaged and coupled portion **145b** is formed for allowing the second pinion gear **122** to be engaged therewith in a unitarily rotatable state.

As illustrated in FIG. 8, the first engaged and coupled portion **145a** is recessed in the inner peripheral part of the disc body **145** that the spool shaft **102** passes therethrough. The first engaging and coupling portion **121c** of the first pinion gear **121** is engaged with the first engaged and coupled portion **145a**. The first engaged and coupled portion **145a** is made in the form of a convexo-concave portion with a diameter slightly smaller than that of the first gear teeth **121a**. The end surface of the first engaging and coupling portion **121c** of the first pinion gear **121** is abutted to a first bottom surface **145d** of the first engaged and coupled portion **145a**.

The second engaged and coupled portion **145b** is recessed on the radially outward of the first engaged and coupled portion **145a**. The second engaging and coupling portion **122b** of the second pinion gear **122** is engaged with the second engaged and coupled portion **145b**. The second engaged and coupled portion **145b** is made in the form of a convexo-concave portion with a diameter slightly smaller than that of the second gear teeth **122a**. The end surface of the second engaging and coupling portion **122b** of the second

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pinion gear **122** is abutted to a second bottom surface **145e** of the second engaged and coupled portion **145b**.

The first pinion gear **121** is herein positioned while being interposed and held between the inner race of the bearing **116c** and the first bottom surface **145d** of the first engaged and coupled portion **145a**. On the other hand, the second pinion gear **122** is positioned while being interposed and held between the second bottom surface **145e** of the second engaged and coupled portion **145b** and a step produced between the first gear teeth **121a** and the first engaging and coupling portion **121c** of the first pinion gear **121**.

As illustrated in FIG. 7, the outer side of the drag disc **142** is covered with a drag cover **155**. The drag cover **155** is made of, for example, die-cast aluminum in consideration of heat dissipation performance. The drag cover **155** is fixed to the end surface of the right-side flange **103b** of the spool **103** by bolt members. A seal member **156** is attached between the drag disc **142** and the inner peripheral side of the drag cover **155**, while a seal member **157** is attached between the drag cover **155** and the spool **103**.

As illustrated in FIG. 7, the moving mechanism **143** includes a drag lever **160**, a pull mechanism **161** and a second spring member **117b**. The drag lever **160** is pivotally mounted to the reel unit **101**. The pull mechanism **161** is configured to pull and move the spool shaft **102** rightwards (in FIG. 7) in conjunction with one-directional pivot of the drag lever **160**. The second spring member **117b** is configured to move the spool shaft **102** leftwards (in FIG. 7) in conjunction with the other-directional pivot of the drag lever **160** by urging the spool shaft **102** leftwards (in FIG. 7).

Actions of Lever Drag Reel

In the lever drag reel structured as described above, the drag lever **160** is pivoted in regulating the magnitude of the drag force of the lever drag mechanism **109**. When the drag lever **160** is set in a drag release position (i.e., the nearest-side pivot position), the friction disc **141** is separated away from the drag disc **142** in the lever drag mechanism **109**. A drag release state is thereby produced, and the spool **103** becomes freely rotatable. Casting can be performed under the condition. When the drag lever **160** is pivoted and operated therefrom towards the away side, the friction disc **141** is gradually moved outwards in the spool shaft direction (i.e., rightwards in FIG. 6). Accordingly, the spool shaft **102** and the spool **103** are gradually moved rightwards. Consequently, the press-contact force of the friction disc **141** onto the drag disc **142** is increased, and thus, the drag force is increased.

When a force greater than a set drag force acts on the fishing line while in fishing, the spool **103** is rotated in the fishing-line releasing direction. At this time, the friction disc **141** is rotated relatively to the drag disc **142**, and accordingly, the drag disc **142** tries to rotate in the fishing-line releasing direction. In response, the anti-reverse mechanism **107** is activated and the claw member **131** is engaged with the ratchet wheel **130**. The drag disc **142** is thereby prevented from reversely rotating. At this time, the claw member **131** is engaged with the ratchet wheel **130** not with the second pinion gear **122**. Therefore, the second pinion gear **122** can be prevented from being damaged or broken. In addition, reverse rotation is prevented at the ratchet wheel **130** in drag activation. Therefore, power is not transmitted from either the first pinion gear **121** or the second pinion gear **122** to either the first drive gear **118** or the second drive gear **119**. Due to the above, the first pinion gear **121** and the second pinion gear **122** are not damaged or broken during drag activation.

Features

The aforementioned exemplary embodiments can be expressed as follows.

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(A) The ratchet wheel **60** (or **130**) for a fishing reel is mounted to the spool shaft **8** (or the drag disc **142**) as the first component for the fishing reel. The ratchet wheel **60** (or **130**) is engaged with the claw portion **62a** (or **131a**) of the claw member **62** (or **131**) pivotally mounted to the spool **4** (or the reel unit **101**) as the second component configured to be rotatable relatively to the first component. The ratchet wheel **60** (or **130**) for the fishing reel includes the mounting part **60a** (or **130a**) and the teeth **60b** (or **130b**). The mounting part **60a** (or **130a**) is mountable to the first component. The teeth **60b** (or **130b**) are radially disposed on the outer peripheral portion **60d** (**130d**) of the mounting part **60a** (or **130a**). The teeth **60b** (or **130b**) are bent from the outer peripheral portion **60d** (**130d**) and extend along the axis **X1** of the spool shaft **8** (or the axis **X2** of the spool shaft **102**) as the rotary axis of the first (or the second) component. The teeth **60b** (or **130b**) allow the claw portion **62a** (or **131a**) of the claw member **62** (or **131**) to be engaged with any adjacent two thereof.

In the ratchet wheel **60** (or **130**) for the fishing reel, the claw portion **62a** (or **131a**) of the claw member **62** (or **131**) is engaged with any adjacent two of the teeth **60b** (or **130b**). The teeth **60b** (or **130b**) extend along the axis **X1** (or **X2**). Therefore, even when the ratchet wheel **60** (or **130**) and the claw member **62** (or **131**) are displaced from each other in the rotary axial direction, the claw portion **62a** (or **131a**) of the claw member **62** (or **131**) is allowed to be engaged with the teeth **60b** (or **130b**). Here, the teeth **60b** (or **130b**) of the ratchet wheel **60** (or **130**) are formed so as to extend from the mounting part **60a** (or **130a**) along the direction of the axis **X1** (or **X2**). Therefore, the length of the teeth **60b** (or **130b**) along the axial direction becomes larger than the thickness of the mounting part **60a** (or **130a**). Accordingly, troubles become unlikely to be caused without elongating the axial length of the claw member **62** (or **131**) even when the ratchet wheel **60** (or **130**) and the claw member **62** (or **131**) are displaced from each other.

(B) The mounting part **60a** (or **130a**) and the teeth **60b** (or **130b**) can be integrally formed by a plate-shaped member and the teeth **60b** (or **130b**) can be bent by pressing. In this case, the ratchet wheel **60** (or **130**) can be easily formed, for example, by producing a circular part to be obtained as the mounting part **60a** (or **130a**) and radial portions to be obtained as the teeth **60b** (or **130b**) disposed on the outer (or inner) peripheral portion of the circular part to be obtained as the mounting part **60a** (or **130a**) by punching, and then, by bending the radial portions to be obtained as the teeth **60b** (or **130b**) by pressing.

(C) The teeth **60b** (or **130b**) can be bent from the outer peripheral portion **60d** (or **130d**) of the mounting part **60a** (or **130a**). In this case, the teeth **60b** (or **130b**) are formed on the outer peripheral portion **60d** (or **130d**) of the mounting part **60a** (or **130a**). Therefore, the portions to be obtained as the teeth **60b** (or **130b**) extend such that the gaps therebetween gradually extend radially outward. Therefore, no limitation is imposed on the length of the teeth **60b** (or **130b**) along the axial direction. Thus, the mounting part **60a** (or **130a**) and the teeth **60b** (or **130b**) can be easily formed by punching.

(D) The teeth can be bent from the inner peripheral portion of the mounting part **60a** (or **130a**). In this case, a limitation is imposed on the length of the teeth along the axial direction when the portions to be obtained as the teeth are formed by punching. However, the ratchet wheel **60** (or **130**) can be formed with a large diameter.

(E) The first component can be the spool shaft **8** for a spinning reel as a fishing reel, while the second component can be the spool **4** rotatably mounted onto the spool shaft **8**. The mounting part **60a** can be non-rotatably mounted onto

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the spool shaft **8**, and is used for the spool sound producing mechanism **20** configured to produce a sound in conjunction with rotation of the spool **4**. In this case, troubles become unlikely to be caused in the spool sound producing mechanism **20** of the spinning reel, even when back-and-front adjustment is performed for the spool **4**.

(F) The first component can be the drag disc **142** configured to be rotated in conjunction with rotation of the handle **104** of a dual-bearing reel as a fishing reel, while the second component can be the reel unit **101** of the dual-bearing reel. The mounting part **130a** can be mounted to the drag disc **142** in a unitarily rotatable state, and can be used for the anti-reverse mechanism **107** configured to prevent the drag disc **142** from rotating in one direction. In this case, troubles become unlikely to be caused in the anti-reverse mechanism **107** of the dual-bearing reel, even when the thickness of the claw member **131** is increased in accordance with drag force.

Other Exemplary Embodiments

Exemplary embodiments of the present invention have been explained above. However, the present invention is not limited to the aforementioned exemplary embodiments, and a variety of changes can be made without departing from the scope of the present invention. Especially, a plurality of exemplary embodiments and modifications described in the present specification can be arbitrarily combined on an as-needed basis.

(a) In the aforementioned exemplary embodiments, the teeth **60b** (or **130b**) are radially disposed on the outer peripheral portion **60d** (or **130d**) of the mounting part **60a** (or **130a**) in the ratchet wheel **60** (or **130**). However, in the present invention, the structure of the teeth **60b** (or **130b**) is not limited to the above. Teeth can be radially disposed on the inner peripheral portion of the mounting part **60a** (or **130a**).

(b) In the aforementioned exemplary embodiments, the ratchet wheel **60** (or **130**) is formed by pressing of a metal plate. However, in the present invention, the method of forming the ratchet wheel **60** (or **130**) is not limited to the above. For example, the ratchet wheel **60** (or **130**) can be formed by forming such as sintering or die casting.

(c) In the first exemplary embodiment, the present invention has been applied to the ratchet wheel **60** of the spool sound producing mechanism **20** for a spinning reel of a front-drag type. However, the application target of the present invention is not limited to the above. For example, the present invention can be applied to ratchet wheels of spool sound producing mechanisms for spinning reels of all types (a lever brake type, a rear drag type, a closed face type, etc.).

(d) In the second exemplary embodiment, the present invention has been applied to the ratchet wheel **130** of the anti-reverse mechanism **107** in the dual-bearing reel. However, the application target of the present invention is not limited to the above. For example, the present invention can be applied to the ratchet wheel **108b** of the spool sound producing mechanism **108** of the dual-bearing reel.

What is claimed is:

1. A ratchet wheel for a fishing reel, the ratchet wheel being mounted to a first component of the fishing reel, the ratchet wheel being engaged with a claw portion of a claw member, the claw portion being pivotally mounted to a second component of the fishing reel, the second component being configured to be rotatable relatively to the first component, the ratchet wheel comprising:

a mounting part configured to be mounted to the first component; and

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a plurality of teeth radially disposed on one of an outer peripheral portion of the mounting part and an inner peripheral portion of the mounting part, the teeth extending from the one of the outer peripheral portion and the inner peripheral portion along a rotary axis of either the first component or the second component in an axial direction so as to be offset from an axially facing surface of the mounting part, the teeth being configured such that two adjacent ones of the teeth selectively engage the claw portion of the claw member.

2. The ratchet wheel according to claim 1, wherein the mounting part and the teeth are integrally formed as a one-piece member in which the mounting part is a plate-shaped member and the teeth are bent parts of the mounting part.

3. The ratchet wheel according to claim 2, wherein the teeth extend from the outer peripheral portion of the mounting part.

4. The ratchet wheel according to claim 1, wherein the teeth extend from the outer peripheral portion of the mounting part.

5. A spinning reel including the ratchet wheel according to claim 1, the spinning reel comprising:

a spool shaft as the first component; and

a spool as the second component, the spool being rotatably mounted onto the spool shaft,

the mounting part being non-rotatably mounted onto the spool shaft and arranged to form a spool sound producing mechanism that is configured to produce a sound in conjunction with rotation of the spool.

6. The spinning reel according to claim 5, wherein the mounting part and the teeth are integrally formed as a one-piece member in which the mounting part is a plate-shaped member and the teeth are bent parts of the mounting part.

7. The spinning reel according to claim 6, wherein the teeth extend from the outer peripheral portion of the mounting part.

8. The spinning reel according to claim 5, wherein the teeth extend from the outer peripheral portion of the mounting part.

9. A dual-bearing reel including the ratchet wheel according to claim 1, the dual-bearing reel comprising:

a handle;

a rotary member as the first component, the rotary member being operatively connected to the handle so that the rotary member rotates in conjunction with rotation of the handle; and

a reel unit as the second component, and

the mounting part being mounted to the rotary member in a unitarily rotatable state and arranged to form an anti-reverse mechanism that prevents the rotary member from rotating in one rotational direction.

10. The dual-bearing reel according to claim 9, wherein the mounting part and the teeth are integrally formed as a one-piece member in which the mounting part is a plate-shaped member and the teeth are bent parts of the mounting part.

11. The dual-bearing reel according to claim 10, wherein the teeth extend from the outer peripheral portion of the mounting part.

12. The dual-bearing reel according to claim 9, wherein the teeth extend from the outer peripheral portion of the mounting part.

13. A ratchet wheel for a fishing reel, the ratchet wheel being mounted to a first component of the fishing reel, the ratchet wheel being engaged with a claw portion of a claw

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member, the claw portion being pivotally mounted to a second component of the fishing reel, the second component being configured to be rotatable relatively to the first component, the ratchet wheel comprising:

a mounting part configured to be mounted to the first component; and

a plurality of teeth radially disposed on an inner peripheral portion of the mounting part, the teeth extending from the inner peripheral portion of the mounting part along a rotary axis of either the first component or the second component, the teeth being configured such that two adjacent ones of the teeth selectively engage the claw portion of the claw member.

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